

USING TOPOLOGY OPTIMIZATION IN LIGHTWEIGHT DESIGN OF FATIGUE RESISTANT STRUCTURES

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Ease in application design methodology of lightweight and fatigue resistant structures is important in many industrial applications. In this article, a proposal of methodology of the fatigue topology optimization is presented. The Dang Van [1] multi-axial high-cycle fatigue and commonly used von Mises constraints are examined using the constant criterion surface algorithm (CCSA) [2]. The CSSA algorithm does not assume a priori the volume value of optimized structure. The algorithm gives possibility to do optimization with various constraints as well as to solve multi-constraint problems.

The Dang Van hypothesis is based on average stresses in an elementary volume V , it considers the average value of shear and normal stresses in this volume. According to Dang Van the fatigue damage appears in a definite time, when the combination of local shear stresses $\tau(t)$ and a hydrostatic stress $\sigma_H(t)$ cuts the borders of an admissible fatigue area.

The numerical convenient form of criterion is as follows [1]:

$$\max_A [\tau(t) + \kappa \sigma_H(t)] \leq \lambda$$

where: A is the area of studied object, $\tau(t) = \frac{\sigma_1(t) - \sigma_3(t)}{2}$, $\sigma_H(t) = \frac{1}{3}(\sigma_1(t) + \sigma_2(t) + \sigma_3(t))$.

The material parameters can be expressed by data from two high-cycle fatigue tests: reversed bending (fatigue limit f_{-1}) and reversed torsion (fatigue limit t_{-1}). For the criterion $\lambda = t_{-1}$, $\kappa = 3 t_{-1} / f_{-1} - 3/2$.

To show clear differences in influence of the von Mises and Dang Van criteria, the Michell's problem of optimizing truss topology with stress constraints under static and variable load conditions was selected as an optimization example [3] (Fig. 1a-b.). The design domain of numerical example is discretized with 300×80 4-node plane stress elements (Young's modulus $E = 210$ GPa, Poisson's ratio $\nu = 0.3$). The static ($F_1 = 100$ N and $F_2 = 200$ N) and pulsating (amplitude $A_1 = 100$ N and $A_2 = 200$ N, $R = 0$) load conditions were applied in four tests to the chosen structure. In the test, the structures volume are minimized with the von Mises static ($\bar{\sigma}_{vM} = 205$ MPa) and Dang Van fatigue stress constraints ($\bar{\sigma}_{DV} = 114$ MPa (assumed material model ($f_{-1} = 190$ MPa, $t_{-1} = 114$ MPa))).

The results of topology optimization are presented in Fig. 1c-f.

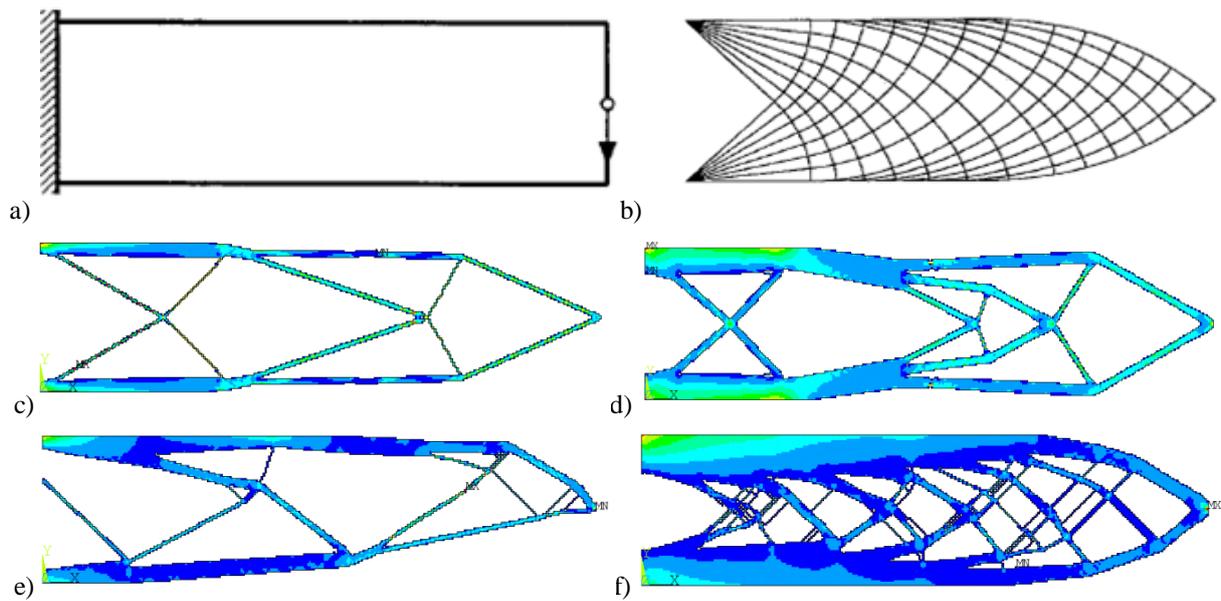


Figure 1: Numerical example: problem description (design domain = 300×80 el. ($V_0=100\%$)) (a), analytical solution (b); numerical solutions for load: $F_1=100$ N ($V_1=15.26\%$, $\sigma_{\max 1}=182$ MPa(von Mises)) (c), $F_2=200$ N ($V_2=31.41\%$, $\sigma_{\max 2}=204$ MPa(von Mises)) (d); numerical solutions for pulsating load of amplitude: $A_1=100$ N ($V_3=22.8\%$, $\sigma_{\max 3}=114$ MPa(Dang Van)) (e), $A_2=200$ N ($V_4=52.43\%$, $\sigma_{\max 4}=105$ MPa(Dang Van)) (f)

One can conclude from presented experiments, that von Mises and Dang Van high-cycle fatigue constraints give different results in optimized structures. It can be clearly seen that the Dang Van criterion discloses the effect of shear stresses in the structure. Moreover, the differences among the volume values ($V_1=15.27\% < V_3=22.8\%$, $V_2=31.41\% < V_4=52.43\%$) indicate the conservative impact of the Dang Van constraints.

In opinion of the author, the method of topology optimization with multiaxial high-cycle fatigue constraints can be recommended for design of lightweight and fatigue resistant structures.

REFERENCES

- [1] Dang Van K., Griveau B., Message O., On a new multiaxial fatigue limit criterion: theory and application, in *Biaxial and Multiaxial Fatigue*, Mechanical Engineering Publications, London, pp. 479–496, 1989.
- [2] Mrzygłód M., Multi-constrained topology optimization using constant criterion surface algorithm, *Bull Polish Academy of Sciences - Tech Sci*, Vol. **60**(2), pp. 229-236, 2012.
- [3] Rozvany G.I.N., Exact analytical solutions for some popular benchmark problems in topology optimization, *Structural Optimization*, Vol. **15**, pp. 42-48, 1998.