

ON THE MODELLING OF TURBULENT FLUID-STRUCTURE INTERACTION. APPLICATION TO A CHANNEL FLOW AROUND A CANTILEVER PLATE ATTACHED BEHIND A CIRCULAR CYLINDER

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Fluid-structure interactions (FSI) arise when the motion of a structure is instigated by a fluid flow and vice-versa, the flow field behavior is affected by the deformation of the structure. This type of coupling can be encountered in a great amount of industrial applications; hence, the understanding of the fluid flow behavior or the awareness of the dynamic action of the structure can be essential to improve engineering designs and optimisations. Therefore, accompanied by a great enhance in computational resources, in the last years the numerical simulation has become an important tool to predict FSI applications.

In order to contribute to the FSI field, the present work attempts the validation of the newly in-house implemented partitioned coupled algorithm focused on the study of the turbulent flow phenomena interacting with a flexible reed type structure. The benchmark case selected consists of the turbulent channel flow around an elastic thin structure clamped behind a fixed rigid non-rotational cylinder proposed in [1]. Several works can be found in the literature which deal with the numerical simulation of this type of application involving laminar flows, but for turbulent flows the sources are still scarce.

The in-house CFD&HT solver [2] is based on a three-dimensional explicit finite volume fractional-step algorithm formulated in a second-order, conservative and collocated unstructured grid arrangement. Large-eddy simulation is used to model the turbulence flow by means of the subgrid scale WALE model [3]. The dynamic action of the structure has been modelled by a specific law according modal analysis of reed theory [4], which provides a fast transient prediction of the solid deformation. Referring to the dynamic mesh procedure, a radial basis function (RBF) interpolation strategy is used [5].

In the final paper the detailed results will be presented and contrasted with the benchmark reference. For instance, in Figure 1 the numerical instantaneous velocity and pressure fields obtained are shown.

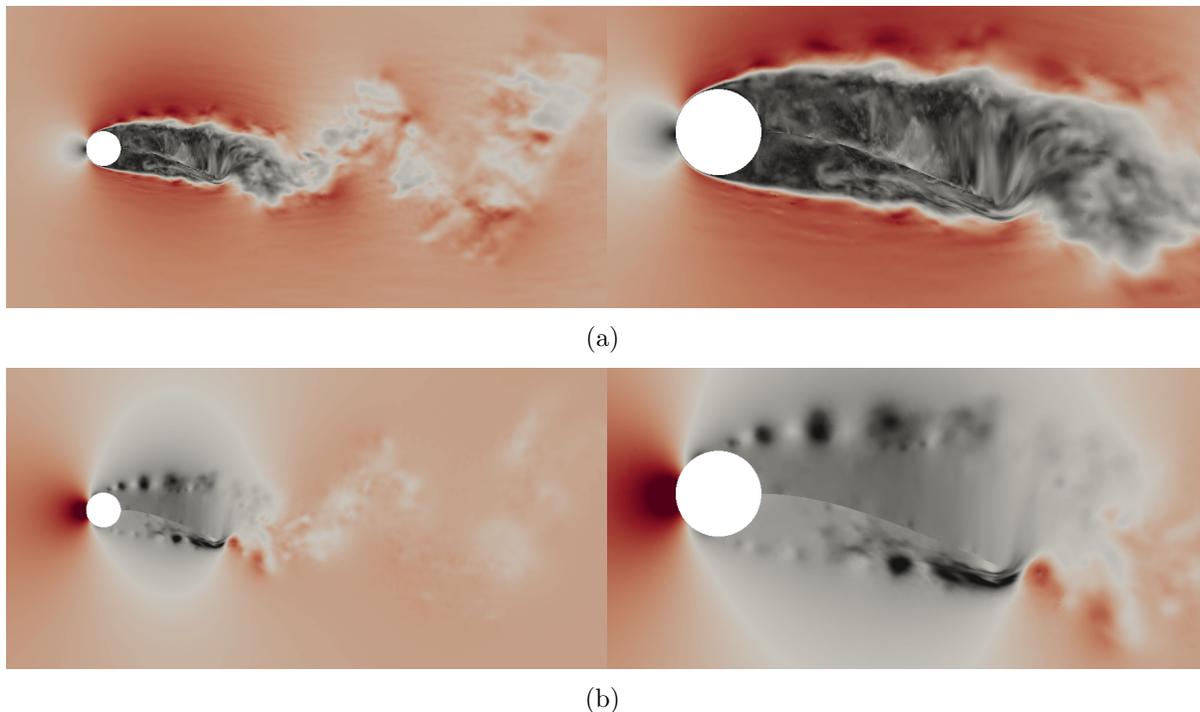


Figure 1: Instantaneous (a) velocity and (b) pressure field.

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