

APPLICATION OF THE PEDE_M TO THE EVALUATION OF RADIATED ACOUSTIC POWER

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The evaluation of the stochastic response of a linear system is a problem which involves several engineering fields. In fact, many dynamic load cases have stochastic behaviour as wall pressure fluctuations due to the turbulent boundary layer (TBL). Nevertheless, the frequency formulation, even when using a discrete coordinate set, can become computationally challenging and this can be only partially mitigated by using a modal representation. A new method named as frequency Modulated Pseudo Equivalent Deterministic Excitation, (PEDE_M), is here applied to calculate the response of a linear elastic system excited by TBL. It is based on the Pseudo Excitation Method, (PEM) which can be considered as an exact and efficient representation of the full stochastic response (FSR) of a linear system. The FSR in term of output displacements is the following

$$\mathbf{S}_{\mathbf{W}\mathbf{W}}(\omega) = \mathbf{\Psi} \mathbf{H}(\omega) \mathbf{\Psi}^T \mathbf{S}_{\mathbf{F}\mathbf{F}}(\omega) \mathbf{\Psi} \mathbf{H}(\omega)^* \mathbf{\Psi}^T \quad (1)$$

being ω , the circular excitation frequency; $\mathbf{S}_{\mathbf{W}\mathbf{W}}$, the output cross-spectral density matrix; \mathbf{H} , the diagonal matrix of the modal mobilities; $\mathbf{\Psi}$, the eigenvectors matrix of the dynamical system; $\mathbf{S}_{\mathbf{F}\mathbf{F}}$, the cross-spectral density matrix of the input load. The superscripts T and $*$ denote the transposition and conjugation of the matrix, respectively. The PEM solution is:

$$\mathbf{S}_{\mathbf{W}\mathbf{W}}(\omega) = \sum_{i=1}^{NG} \mathbf{w}^*(\omega, i) \mathbf{w}^T(\omega, i) \quad (2)$$

$$\mathbf{w}(\omega, i) = \mathbf{\Psi} \mathbf{H}(\omega) \mathbf{\Psi}^T \mathbf{\Theta}^{<i>} \sqrt{d_i(\omega)} \quad (3)$$

$$\mathbf{S}_{\mathbf{FF}}(\omega) = \sum_{i=1}^{NG} d_i(\omega) \boldsymbol{\Theta}^{<i>} \boldsymbol{\Theta}^{<i>T} \quad (4)$$

where $\boldsymbol{\Theta}$ and d are the eigenfunctions of the load matrix; NG is the number of degrees of freedom. Finally, the PEDE_M solution is:

$$\hat{\mathbf{S}}_{\mathbf{WW}}(\omega) = \sum_{i=1}^{NG} \hat{\mathbf{w}}^*(\omega, i) \hat{\mathbf{w}}^T(\omega, i) \quad (5)$$

$$\hat{\mathbf{w}}(\omega, i) = \boldsymbol{\Psi} \mathbf{H}(\omega) \boldsymbol{\Psi}^T \sqrt{\mathbf{S}_{\mathbf{FF}}^{<i>}(\omega)} \quad (6)$$

Thus, PEDE_M tries to overcome the analysis of the eigensolutions of the load matrix in order to reduce the computational cost. This can be done for the present application by analysing the limits of the $\mathbf{S}_{\mathbf{FF}}$ in the *low* and *high* frequency range. In the present work, PEDE_M is applied to evaluate the acoustic radiated power from an elastic surface, Π , for the effect of TBL. Its expression can be written as:

$$\Pi(\omega) = \frac{\omega^4 \rho_a}{4\pi c_a} \mathbf{A}^T \boldsymbol{\Psi} \mathbf{H}(\omega) \boldsymbol{\Psi}^T \mathbf{S}_{\mathbf{FF}}(\omega) \boldsymbol{\Psi} \mathbf{H}(\omega)^* \boldsymbol{\Psi}^T \mathbf{A} \quad (7)$$

where \mathbf{A} is the equivalent nodal area vector; ρ_a and c_a are the fluid density and speed of sound of the fluid, respectively. The results will show that PEDE_M can be very useful to further simplify the solution response.

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